



US009116339B2

(12) **United States Patent**  
**Ruyten**

(10) **Patent No.:** **US 9,116,339 B2**  
(45) **Date of Patent:** **Aug. 25, 2015**

(54) **COMBINER POSITIONING SYSTEM**

(71) Applicant: **Jabil Circuit, Inc.**, Saint Petersburg, FL  
(US)

(72) Inventor: **Henricus Marie Ruyten**,  
Gumpoldskirchen (AT)

(73) Assignee: **Jabil Circuit, Inc.**, St. Petersburg, FL  
(US)

(\*) Notice: Subject to any disclaimer, the term of this  
patent is extended or adjusted under 35  
U.S.C. 154(b) by 305 days.

(21) Appl. No.: **13/655,996**

(22) Filed: **Oct. 19, 2012**

(65) **Prior Publication Data**

US 2013/0100535 A1 Apr. 25, 2013

**Related U.S. Application Data**

(60) Provisional application No. 61/549,991, filed on Oct.  
21, 2011.

(51) **Int. Cl.**

**G02B 27/14** (2006.01)  
**G09G 5/00** (2006.01)  
**G02B 27/01** (2006.01)  
**B60R 11/02** (2006.01)  
**B60K 35/00** (2006.01)  
**B60R 11/00** (2006.01)

(52) **U.S. Cl.**

CPC ..... **G02B 27/0149** (2013.01); **B60K 35/00**  
(2013.01); **B60R 11/0229** (2013.01); **B60K**  
**2350/2052** (2013.01); **B60K 2350/405**  
(2013.01); **B60R 2011/0092** (2013.01); **G02B**  
**2027/0154** (2013.01)

(58) **Field of Classification Search**

USPC ..... 359/630, 632, 636; 345/7  
See application file for complete search history.

(56) **References Cited**

**U.S. PATENT DOCUMENTS**

5,204,666	A	4/1993	Aoki et al.	
7,978,414	B2	7/2011	Potakowskyj et al.	
2002/0167189	A1	11/2002	Nakamura et al.	
2003/0223133	A1 *	12/2003	Raines et al.	359/841
2006/0203351	A1 *	9/2006	Kageyama et al.	359/630
2009/0086329	A1 *	4/2009	Potakowskyj et al.	359/630

**FOREIGN PATENT DOCUMENTS**

EP	2 477 058	7/2012
WO	WO-2007/137996	12/2007

**OTHER PUBLICATIONS**

International Search Report and Written Opinion issued on PCT/  
US2012/061114, mailed Feb. 7, 2013.

\* cited by examiner

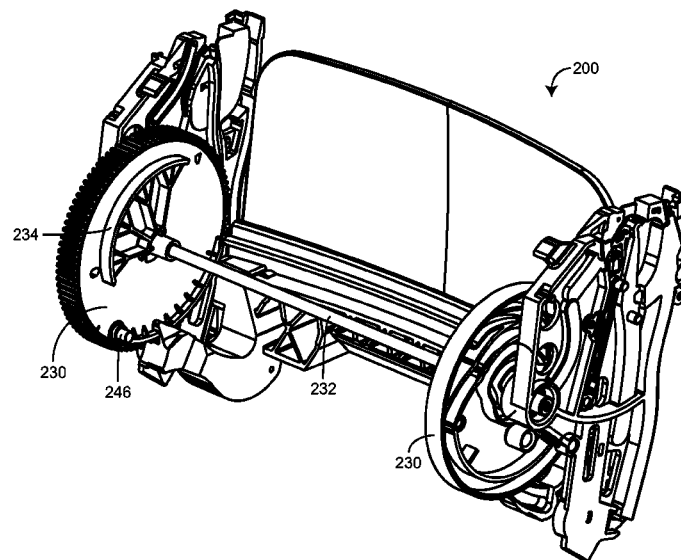
*Primary Examiner* — Alicia M Harrington

(74) *Attorney, Agent, or Firm* — Volpe and Koenig, P.C.

(57) **ABSTRACT**

A combiner positioning system for a head-up display includes a carrier and a camwheel. The carrier supports a combiner for the head-up display, and the combiner is movable between a stowed position and a display position. The camwheel is rotatable about a camwheel axis. The rotation of the camwheel about the camwheel axis engages the carrier and moves the combiner between the stowed position and the display position.

**18 Claims, 17 Drawing Sheets**



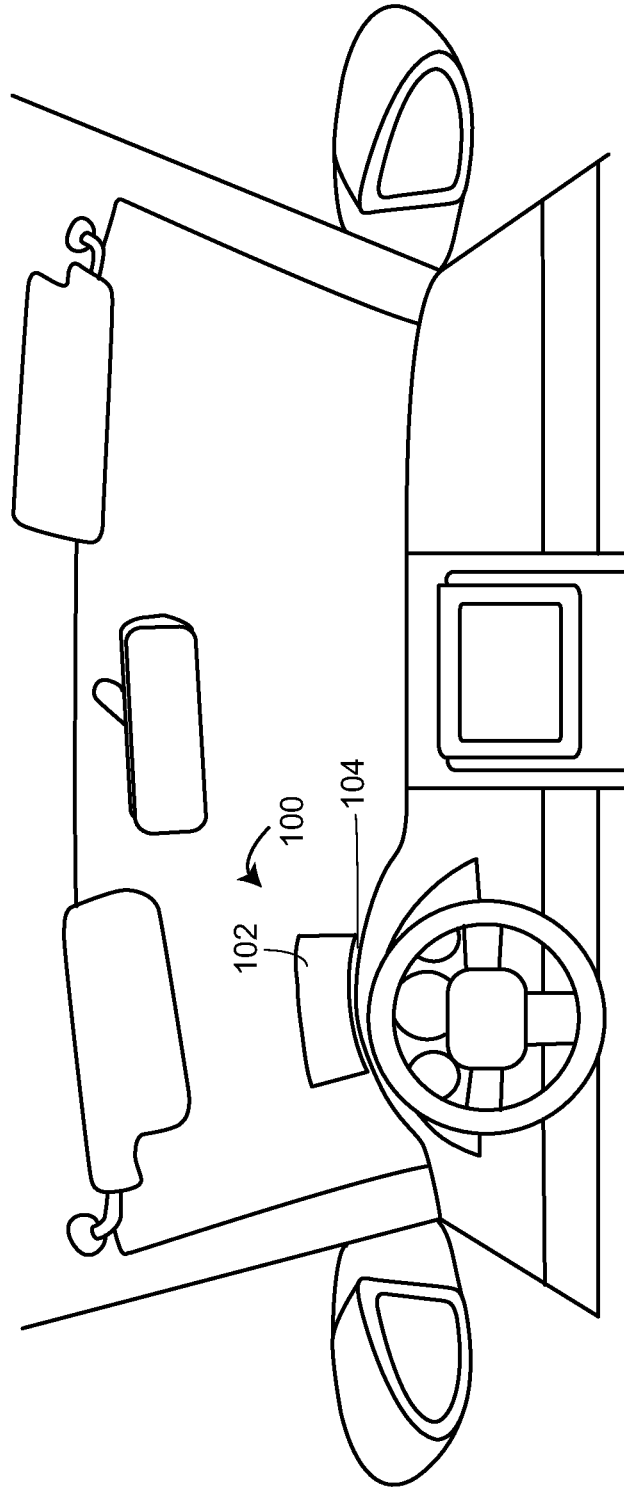


FIG. 1

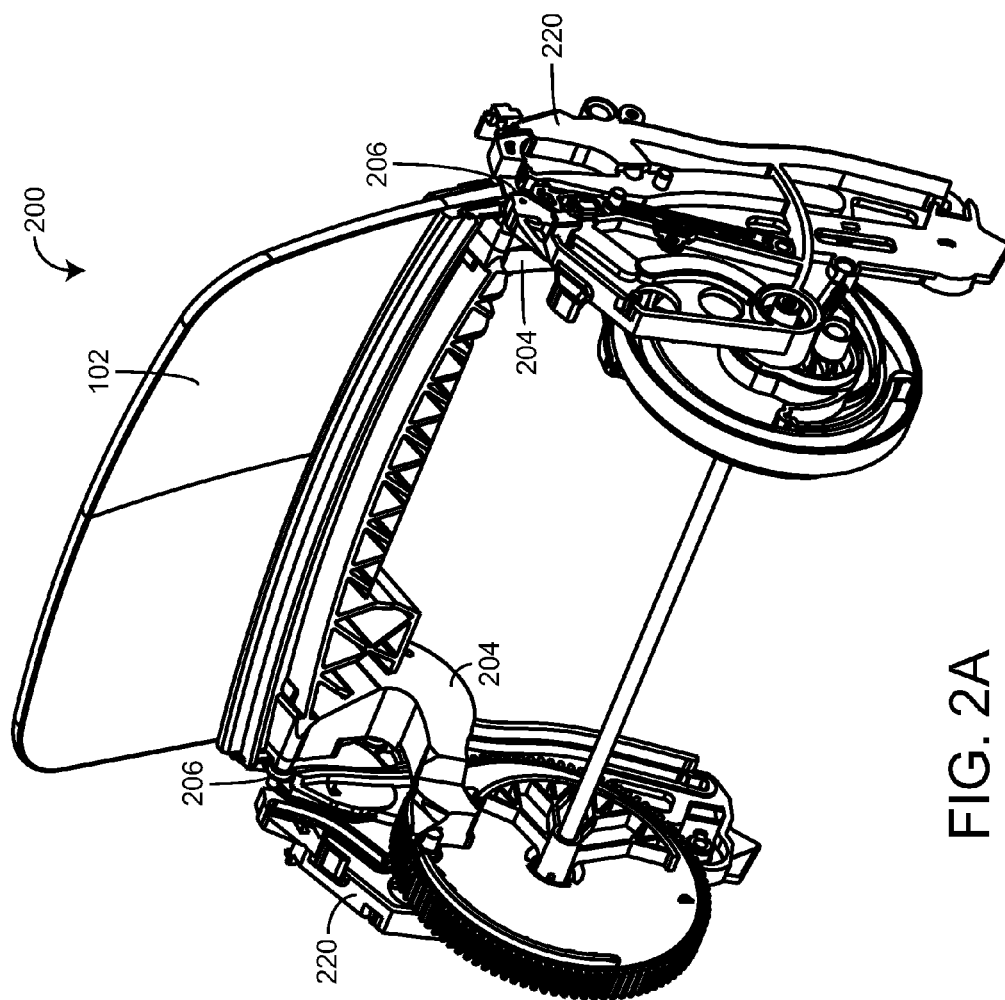


FIG. 2A

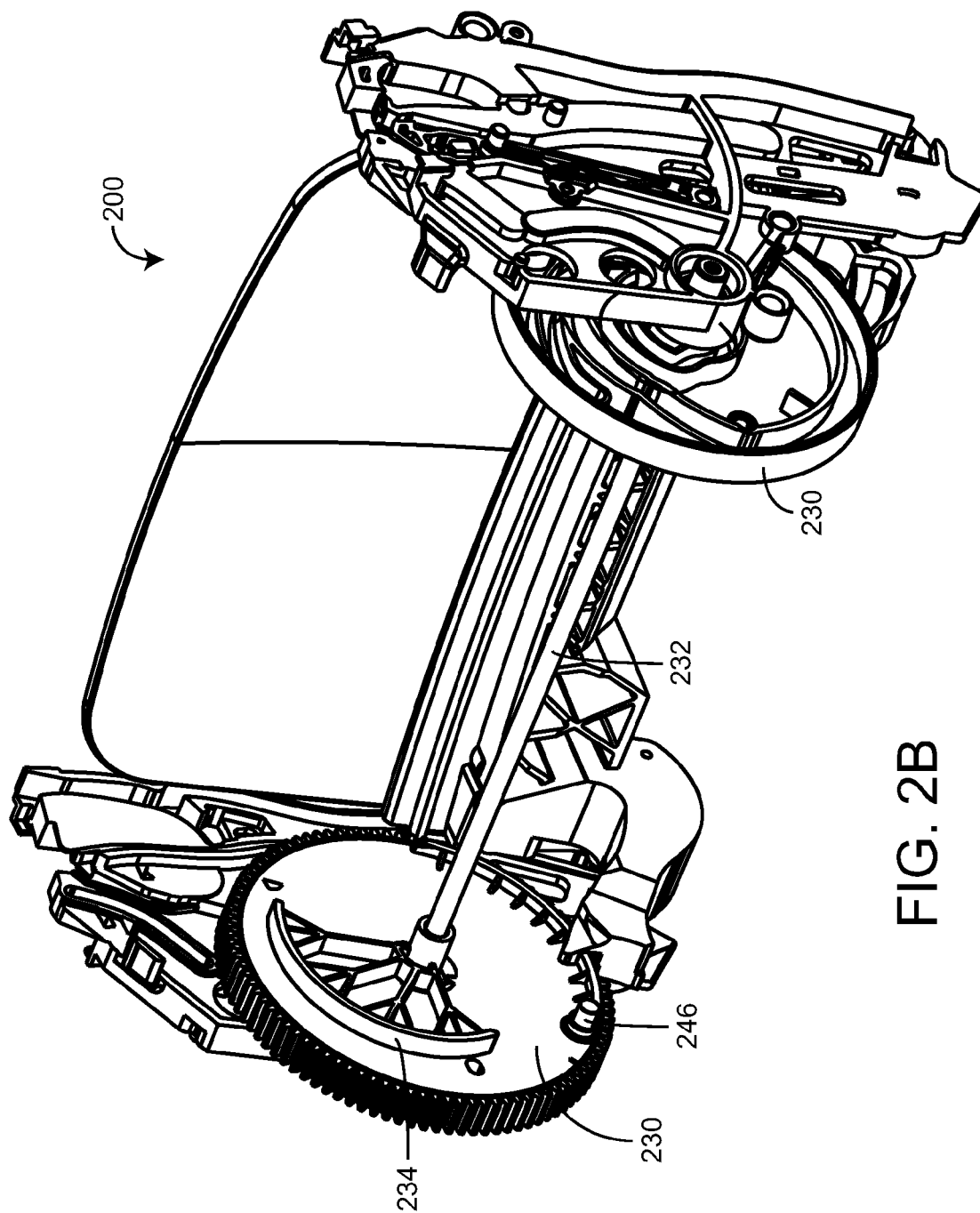


FIG. 2B

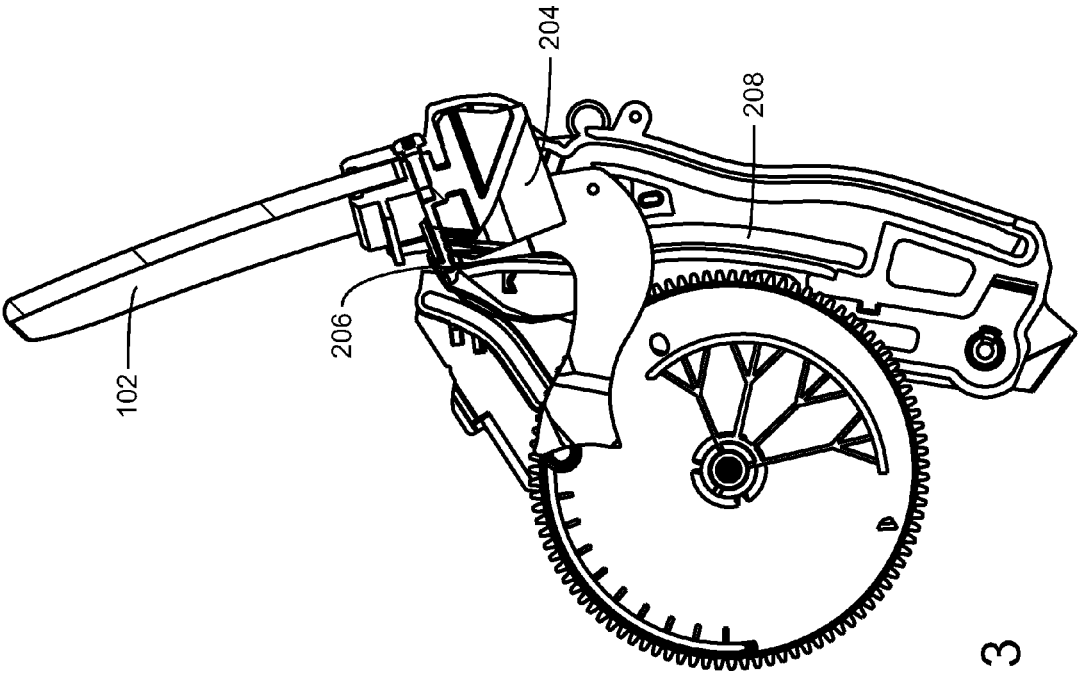


FIG. 3

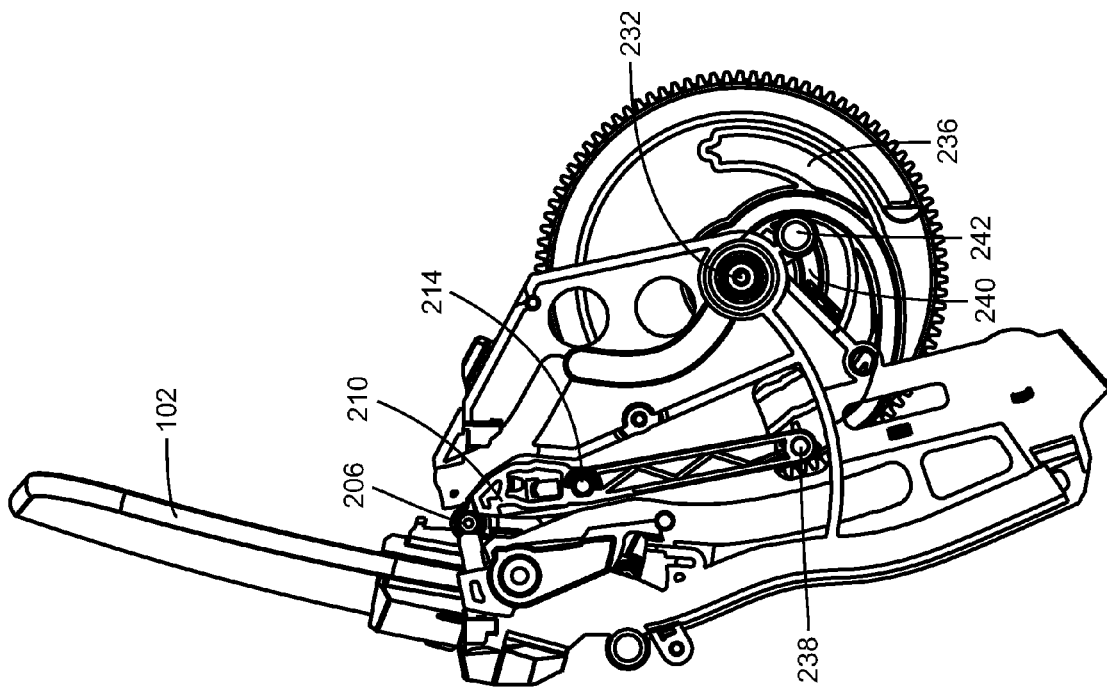


FIG. 4A

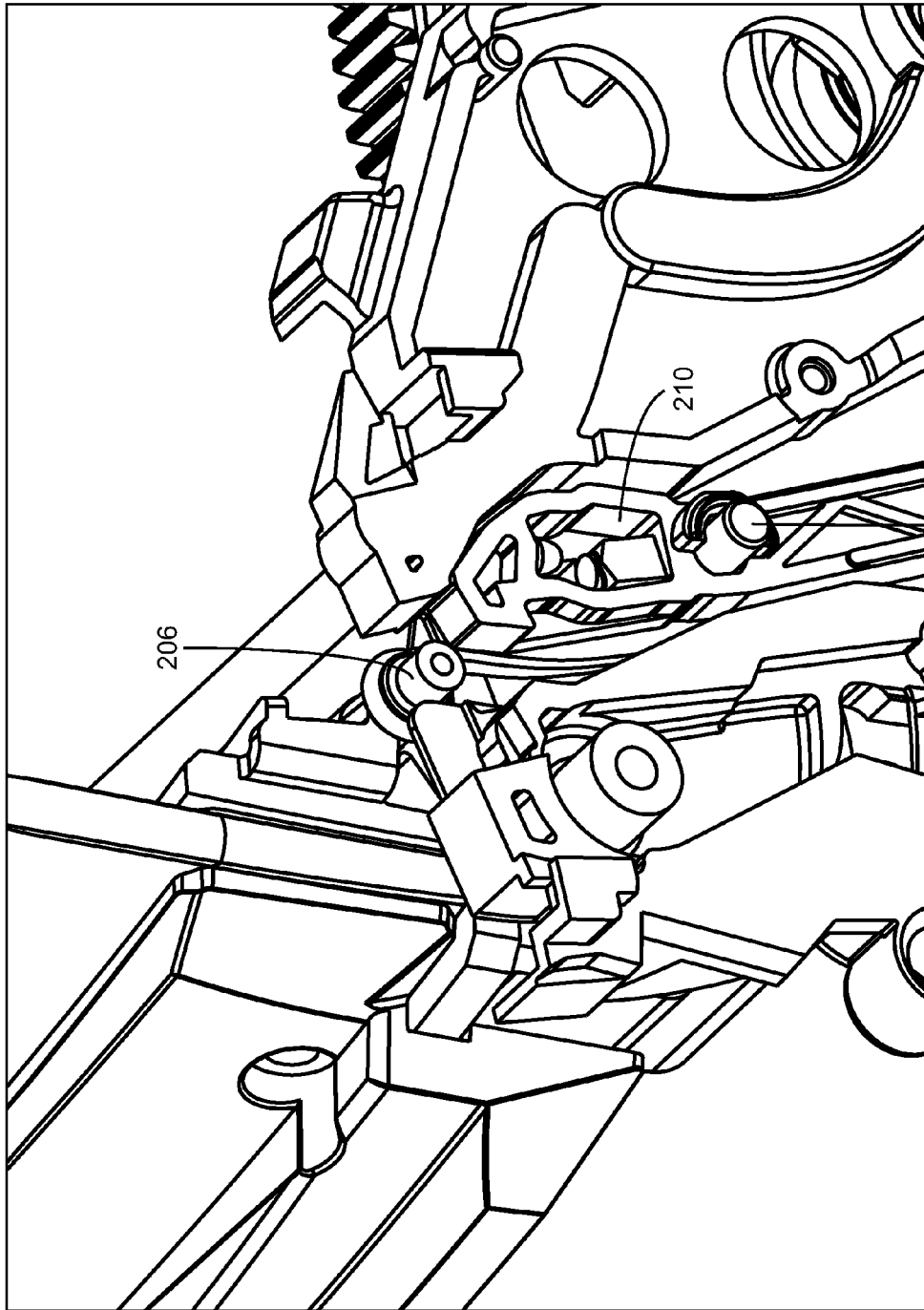


FIG. 4B

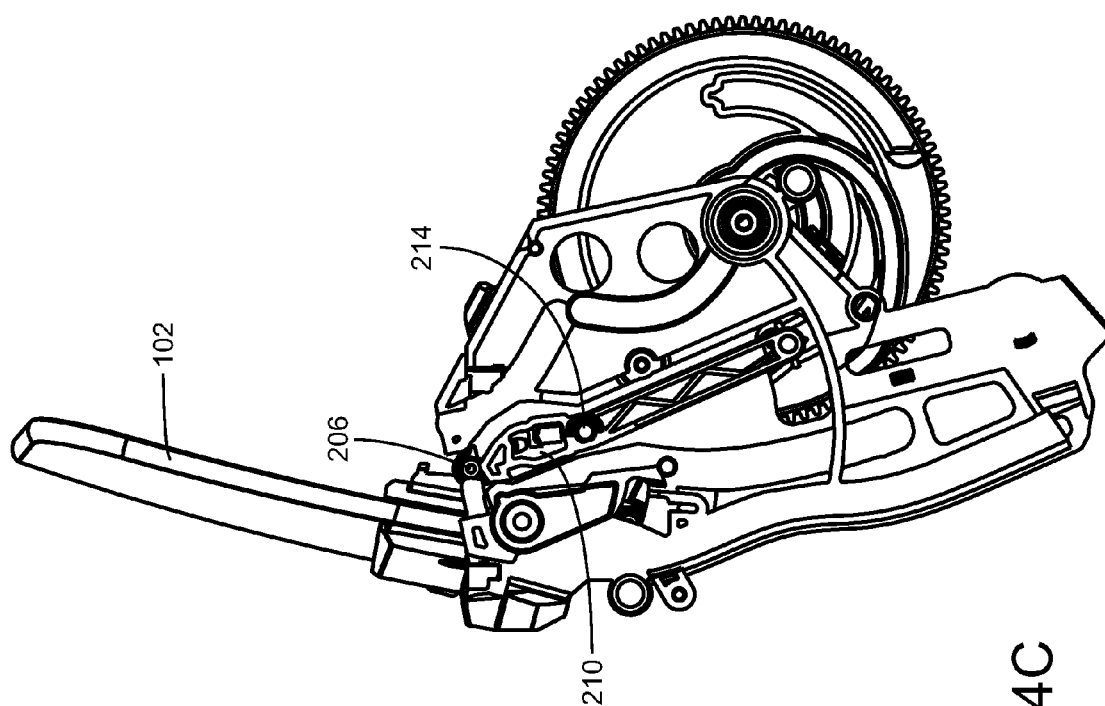


FIG. 4C



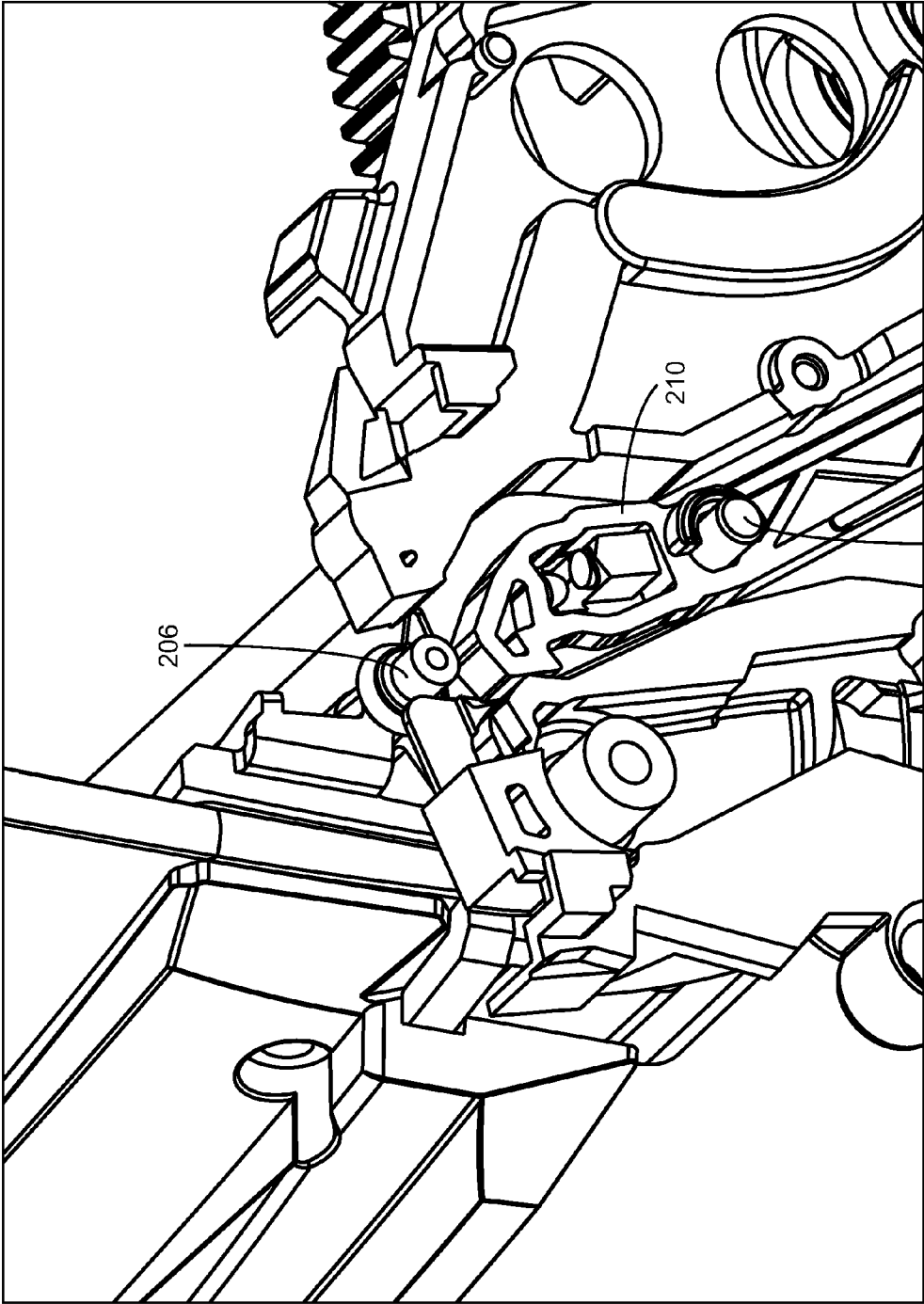


FIG. 4D

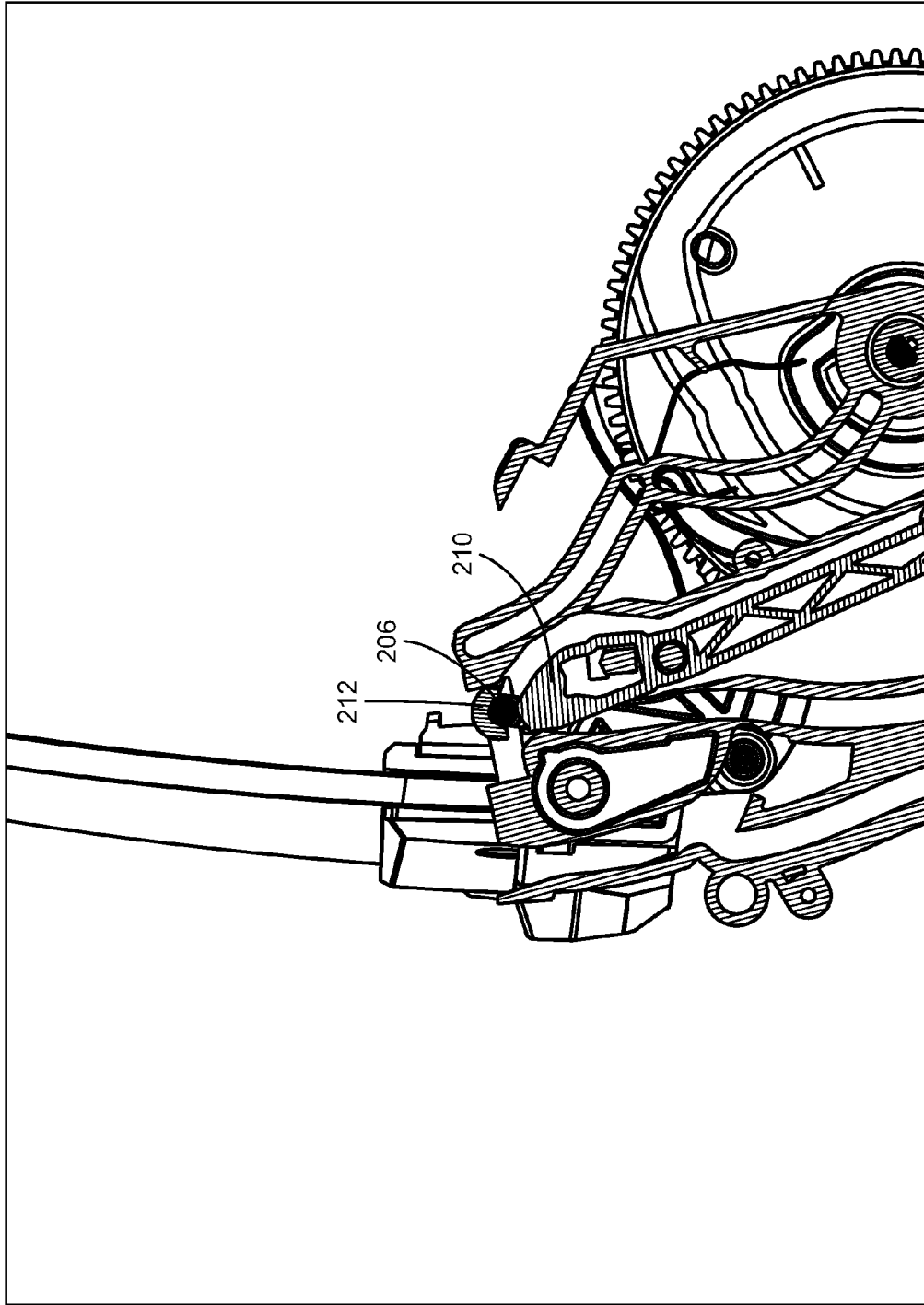


FIG. 4E

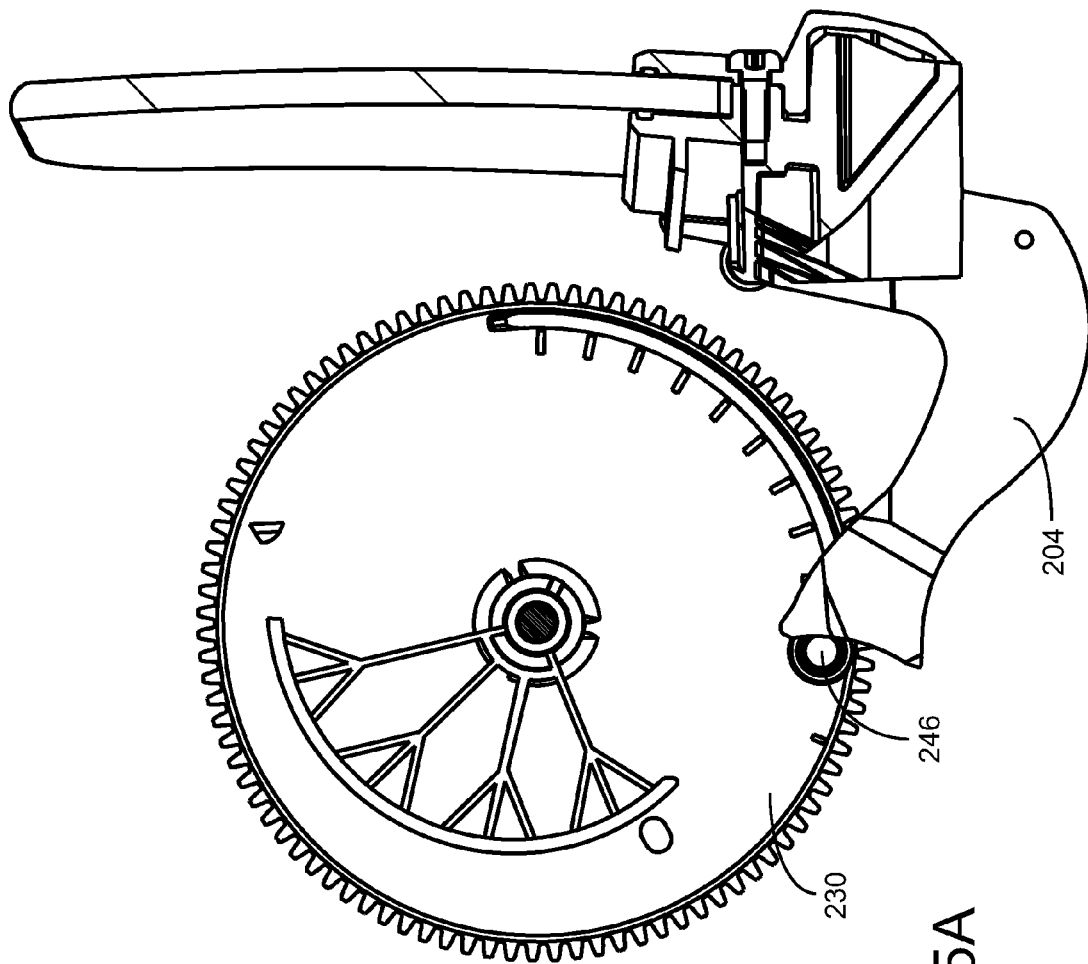


FIG. 5A

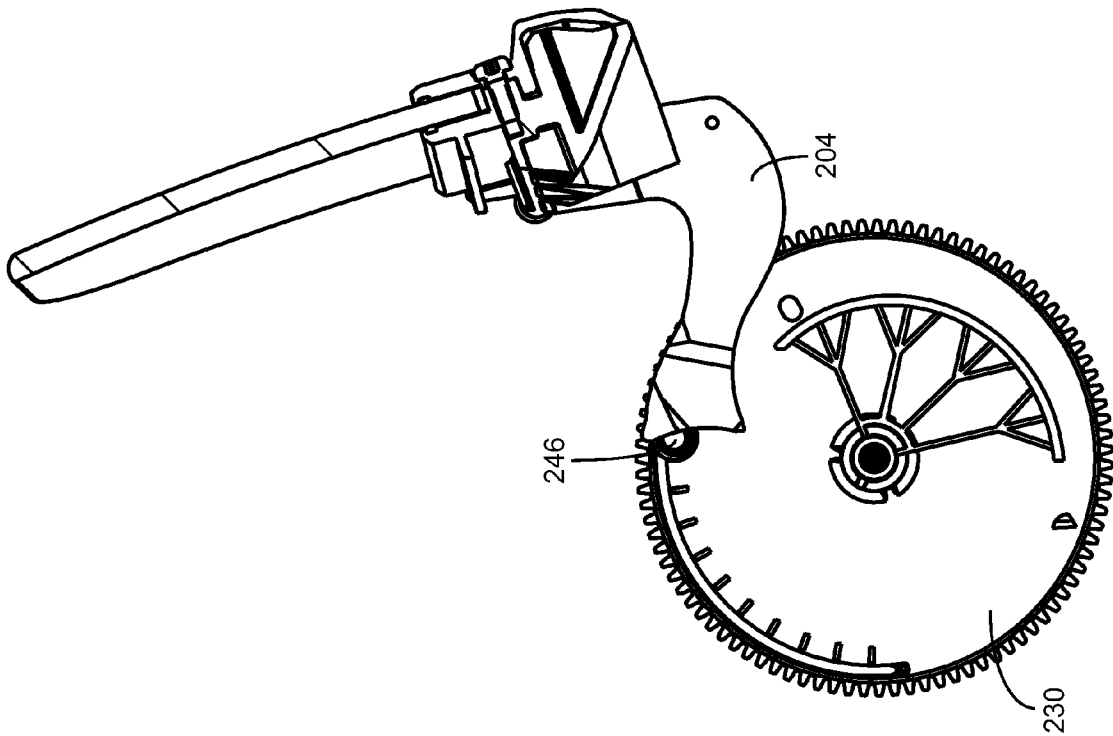


FIG. 5B

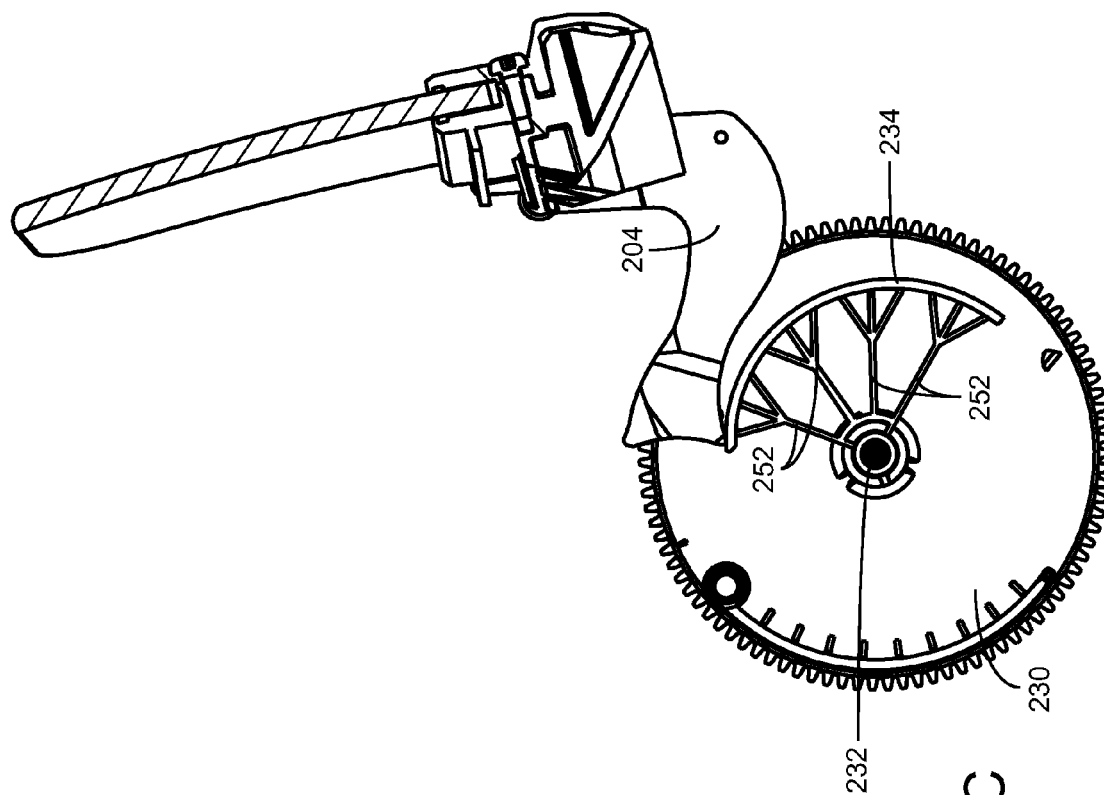


FIG. 5C

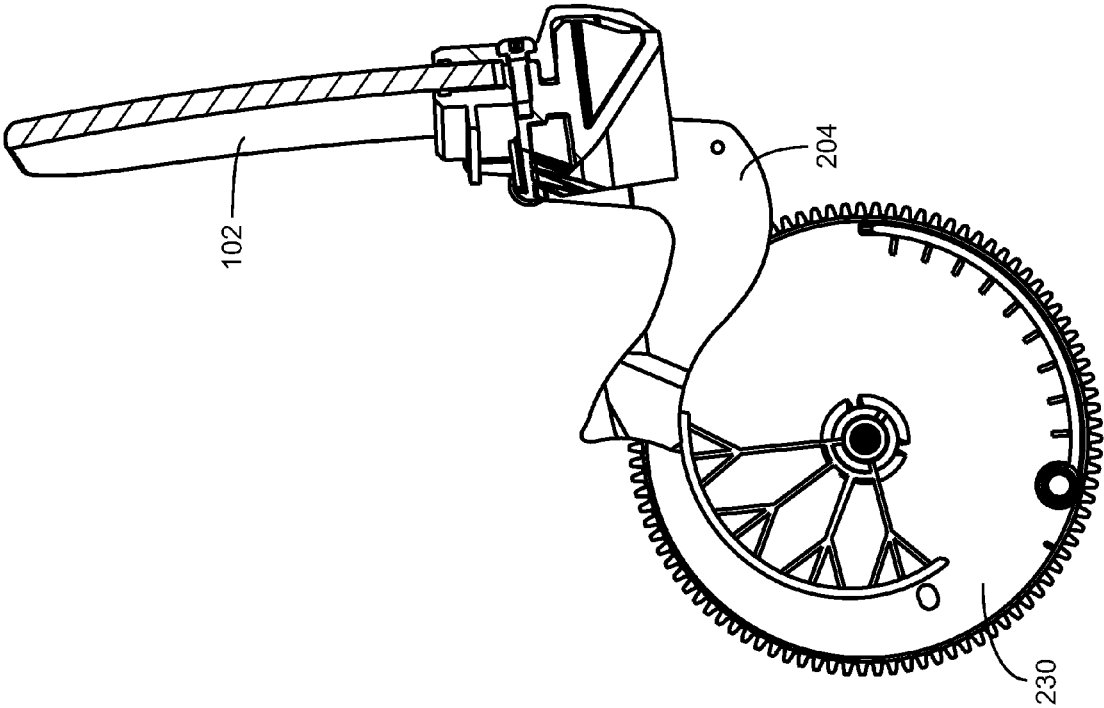


FIG. 5D

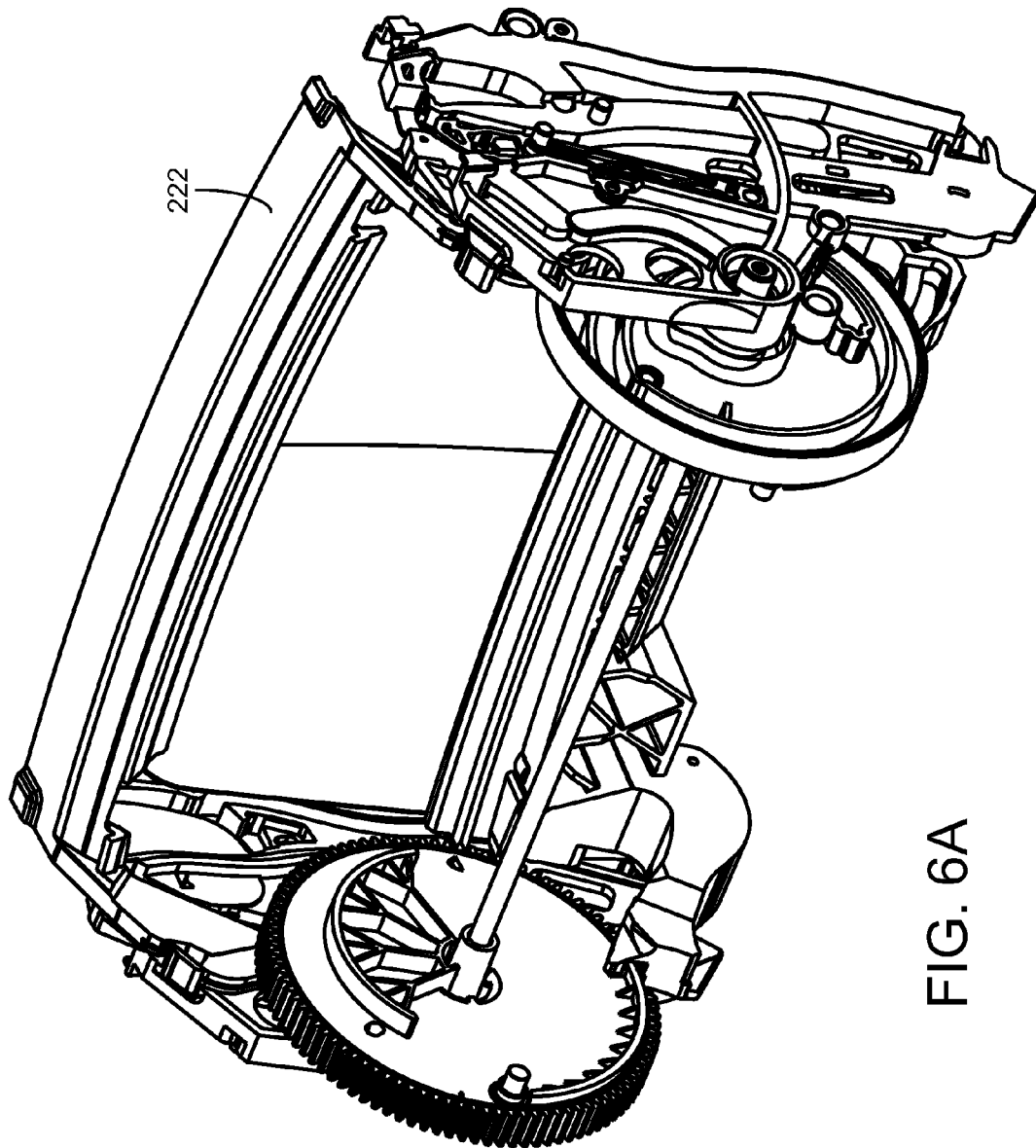


FIG. 6A

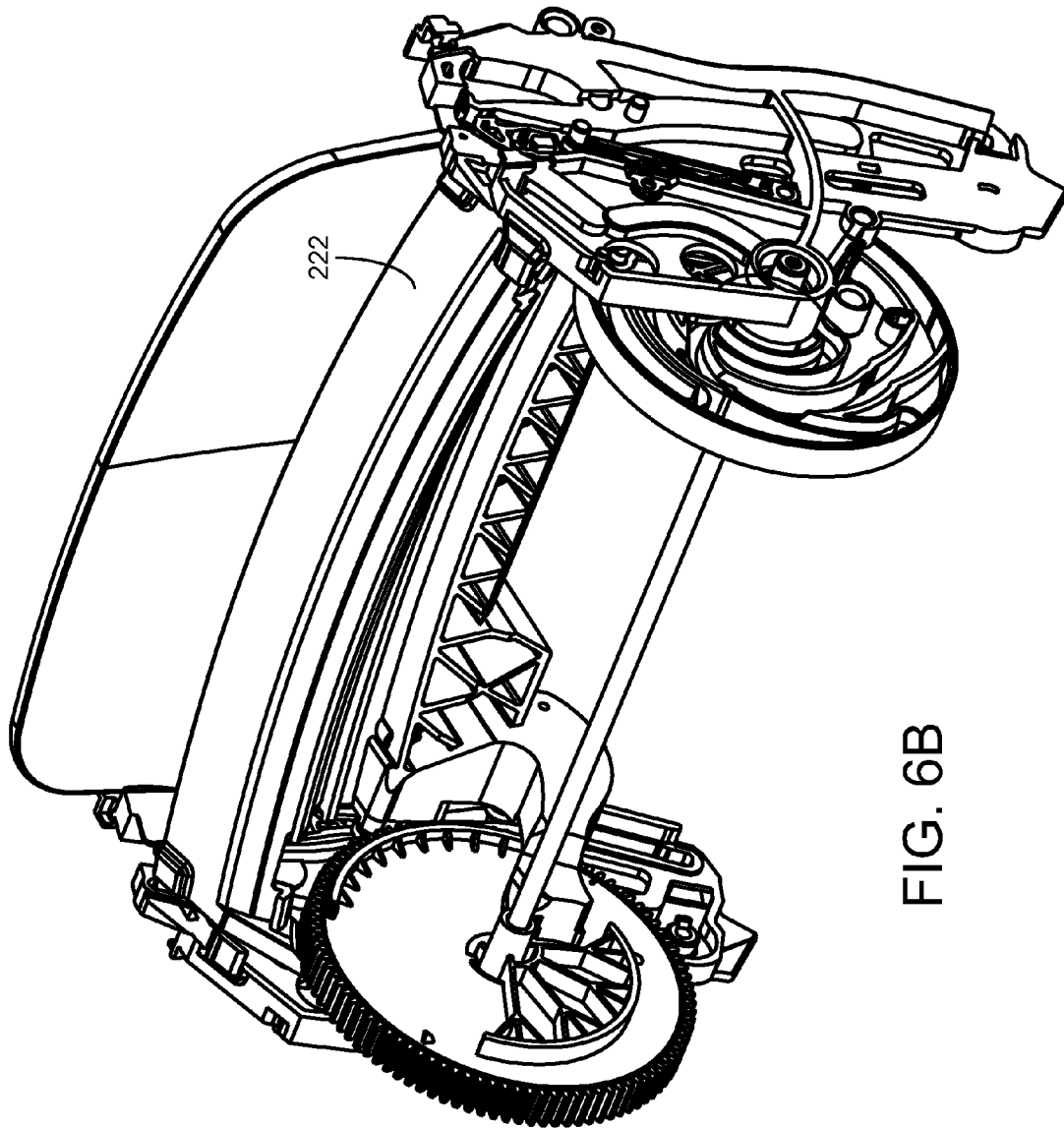


FIG. 6B



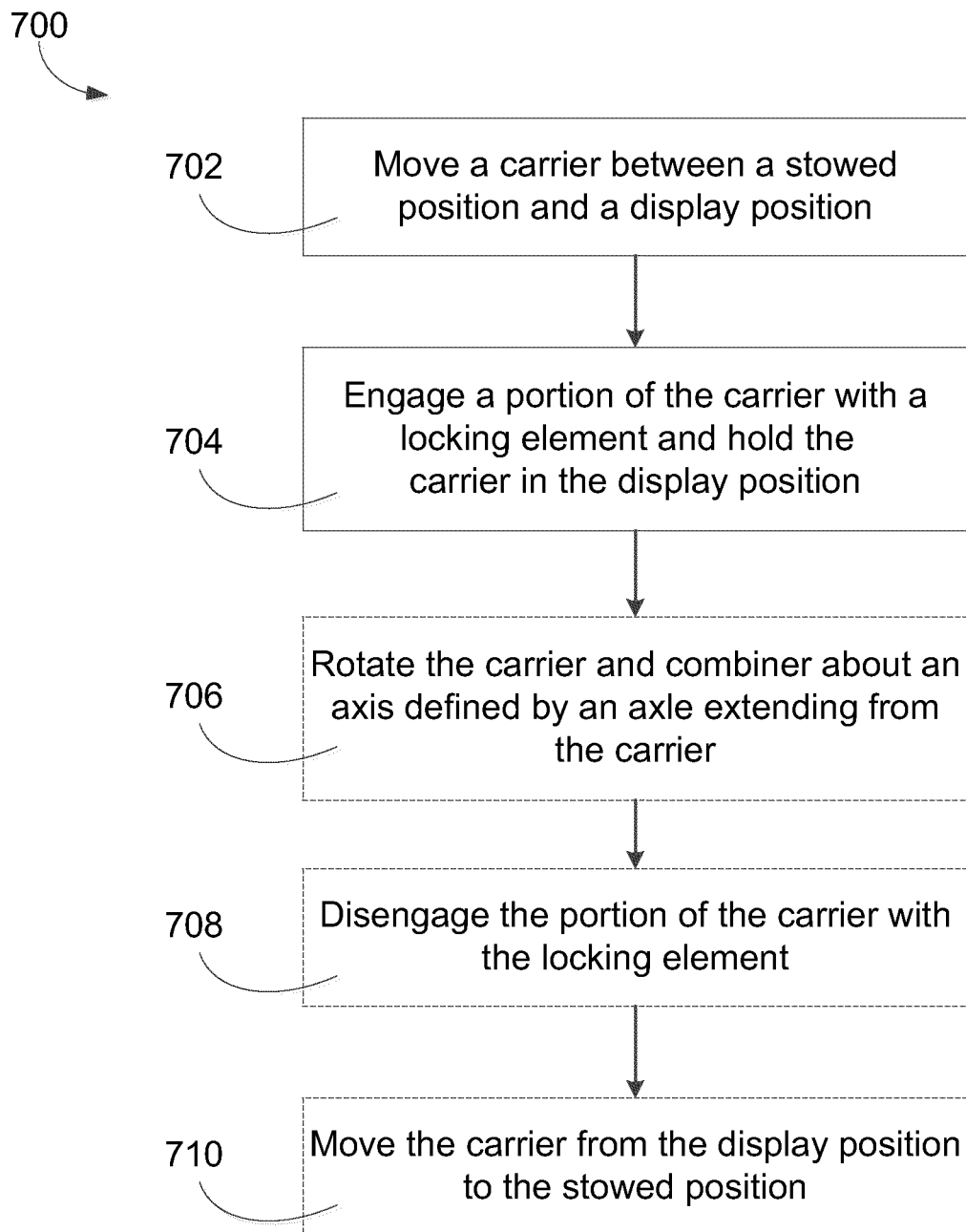
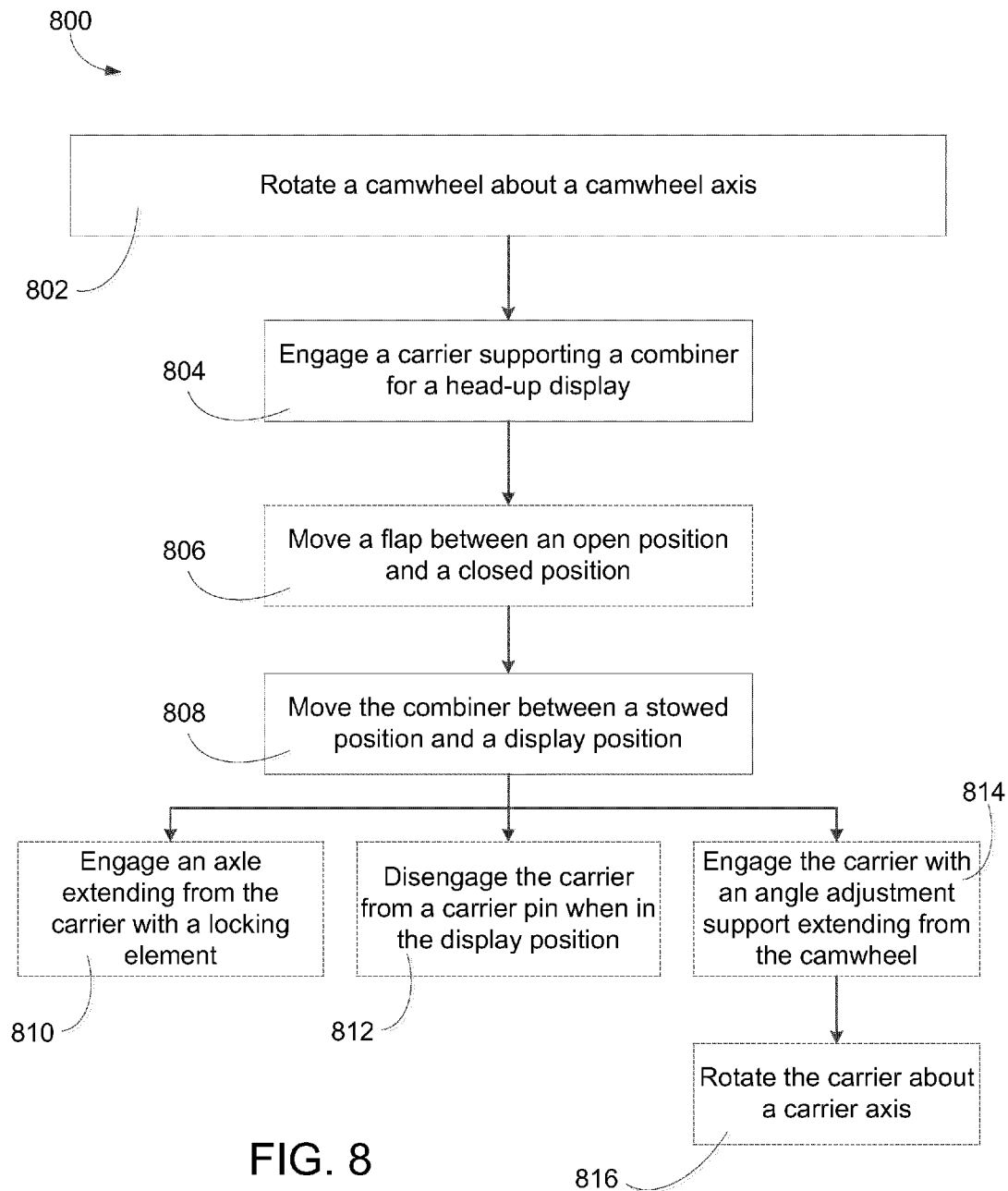


FIG. 7



**COMBINER POSITIONING SYSTEM****CROSS-REFERENCE TO RELATED APPLICATIONS**

The present application claims the benefit of U.S. Provisional Application No. 61/549,991, filed Oct. 21, 2011, the entire contents of which are hereby incorporated by reference.

**BACKGROUND**

A head-up display (HUD) is a translucent display panel which may be used to present information to a user without requiring the user to divert his or her gaze from a preferred gaze direction. For example, a HUD may be used to present a driver of a vehicle with a variety of information (e.g., speed, fuel consumption, navigation information, time, etc.) while allowing the driver to simultaneously view traffic conditions and the road in front of the vehicle.

A typical HUD includes a combiner, a projector, and a visual data source. The combiner is a translucent panel upon which visual data is projected. The combiner usually includes one or more optical coatings that reflect only the specific wavelengths of monochromatic light which are projected by the HUD projector. Thus, the combiner can be conceptualized as a translucent mirror. The HUD projector is positioned relative to the combiner to allow images, graphics, video, or other visual data to be projected onto the combiner. The combiner then reflects the projected images toward the user.

HUD systems require careful design consideration to ensure reliable performance under a variety of operational conditions. For example, a HUD may be exposed to varying environmental conditions such as high humidity, extreme temperatures, and the presence of contaminants which may cause corrosion. Additionally, when a HUD is implemented in a mobile environment, it is desirable to protect the combiner when the HUD system is inactive to mitigate the potential of damaging the combiner. For this purpose, a positioning system may be used to move the combiner between an active display position and a protected storage position.

Repeatable positional accuracy is important for combiner positioning systems because an improperly positioned combiner will not function properly in a HUD system. Such positional accuracy may be achieved by using high precision parts which are manufactured and maintained to very close tolerances. However, the need for such close tolerances has the undesirable effect of increasing the manufacture and maintenance expense associated with the positioning system.

Further, failure to control vibration associated with the combiner may result in a display that is irritating to the user and unpleasant to view over extended periods. Thus, a positioning system is needed which allows the combiner to move between a protected position and a display position and which reduces undesirable vibration to allow for a pleasant viewing experience.

**SUMMARY**

In one embodiment, a combiner positioning system for a head-up display includes a carrier supporting a combiner for the head-up display, wherein the combiner is movable between a stowed position and a display position, and a camwheel rotatable about a camwheel axis, wherein rotation of the camwheel about the camwheel axis engages the carrier and moves the combiner between the stowed position and the

display position. In some embodiments, the camwheel engages the carrier directly without any intermediate components.

In some embodiments, the combiner positioning system further includes an angle adjustment support extending from the camwheel, wherein rotation of the camwheel about the camwheel axis engages the carrier with the angle adjustment support and rotates the combiner about a carrier axis, wherein the angle adjustment support is a self-locking element. In some embodiments, the angle adjustment support rigidly holds the carrier in the display position and increases a stiffness of the combiner positioning system when holding the carrier in the display position.

In some embodiments, the combiner positioning system further includes a locking element having an engaged position and a disengaged position, wherein the locking element engages an axle extending from the carrier in the display position and rigidly holds the axle in the display position. In some embodiments, when the locking element is in the disengaged position, the locking element does not engage the axle and does not restrain the axle from moving laterally, wherein lateral movement of the axle is required for the combiner to move between the stowed position and the display position. In some embodiments, the locking element may be shaped as a lever rotatable about a lever axis between the engaged position and the disengaged position, wherein the locking element is coupled to the camwheel via a pin extending from the locking element and fitted into a slot in the camwheel, wherein rotation of the camwheel about the camwheel axis rotates the locking element about the lever axis between the engaged position and the disengaged position.

In some embodiments, a pin extending from the camwheel disengages from the carrier when in the display position such that further rotation of the camwheel rotates the carrier about a carrier axis but does not cause further lateral movement of the carrier, wherein lateral movement is required for the carrier to move between the stowed position and the display position.

In some embodiments, the combiner positioning system further includes a guide element defining a path along which an axle extending from the carrier moves between the stowed position and the display position or a housing within which the combiner is contained when in the stowed position and from which the combiner protrudes when in the display position.

In some embodiments, the combiner positioning system further includes a flap movable between an open position and a closed position, wherein the flap covers an opening the housing when in the closed position and exposes the opening when in the open position, wherein the flap is coupled to the camwheel via a pin extending from the flap and fitted into a slot in the camwheel, wherein rotation of the camwheel about the camwheel axis moves the flap between the open position and the closed position.

The combiner may be moved between a stowed position and a display position using a method including rotating a camwheel about a camwheel axis, engaging a carrier supporting a combiner for a head-up display, and moving the combiner between a stowed position and a display position, wherein rotation of the camwheel about the camwheel axis engages the carrier and moves the combiner between the stowed position and the display position. In some embodiments, the camwheel engages the carrier directly without any intermediate components.

In some embodiments, the method for moving the combiner further includes engaging the carrier with an angle adjustment support extending from the camwheel, wherein

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the angle adjustment support engages the carrier when in the display position, and rotating the carrier about a carrier axis, wherein further rotation of the camwheel about the camwheel axis engages the carrier with the angle adjustment support and rotates the carrier about the carrier axis via the angle adjustment support. In some embodiments, the method for moving the combiner further includes rigidly holding the carrier in the display position with the angle adjustment support.

In additional embodiments, the method includes engaging an axle extending from the carrier with a locking element, wherein the locking element has an engaged position and a disengaged position, and rigidly holding the axle in the display position with the locking element, wherein when the locking element is in the disengaged position, the locking element does not engage the axle and does not restrain the axle from moving laterally, wherein lateral movement of the axle is required for the combiner to move between the stowed position and the display position. In some embodiments the locking element is shaped as a lever rotatable about a lever axis between the engaged position and the disengaged position, wherein the locking element is coupled to the camwheel such that rotation of the camwheel about the camwheel axis rotates the locking element about the lever axis between the engaged position and the disengaged position.

In some embodiments, the method for moving the combiner further includes disengaging the carrier from a pin extending from the camwheel when in the display position such that further rotation of the camwheel does not cause further lateral movement of the carrier once in the display position, wherein lateral movement is required for the carrier to move between the stowed position and the display position. The combiner may be contained within a housing when in the stowed position and may protrude from the housing when in the display position.

In some embodiments, the method for moving the combiner further includes moving a flap between an open position and a closed position, wherein the flap covers the slot when in the closed position and exposes the slot when in the open position, and wherein the flap is coupled to the camwheel such that rotation of the camwheel about the camwheel axis moves the flap between the open position and the closed position.

### BRIEF DESCRIPTION OF THE DRAWINGS

FIG. 1 is a drawing of a head-up display system implemented in a vehicle and shown in the display position.

FIG. 2A is a drawing showing a combiner positioning system with the combiner in the display position.

FIG. 2B is a drawing showing the combiner positioning system with the combiner in the stowed position.

FIG. 3 is a drawing showing a half-sectional side perspective view of the combiner positioning system with the combiner in the display position.

FIG. 4A is a drawing showing a side view of the combiner positioning system with a locking element in a disengaged position.

FIG. 4B is a drawing showing a close-up view of a carrier axle in the display position, with the locking element in the disengaged position.

FIG. 4C is a drawing showing a side view of the combiner positioning system with a locking element in an engaged position.

FIG. 4D is a drawing showing a close-up view of the carrier axle in the display position, with the locking element in the engaged position.

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FIG. 4E is a drawing showing the carrier axle rigidly held in a recess by the locking element.

FIG. 5A is a drawing showing a half-sectional side perspective view of a camwheel engaging the carrier via a carrier pin extending from the camwheel when the combiner is in the stowed position.

FIG. 5B is a drawing showing a half-sectional side perspective view of the carrier pin disengaging from the camwheel with the combiner in the display position.

FIG. 5C is a drawing showing a half-sectional side perspective view of an angle adjustment support engaging the carrier with the combiner in the display position.

FIG. 5D is a drawing showing a half-sectional side perspective view of the angle adjustment support supporting the carrier in the display position with the combiner fully rotated.

FIG. 6A is a drawing of the combiner positioning system showing a flap in a closed position, covering the combiner in the stowed position.

FIG. 6B is a drawing of the combiner positioning system showing the flap in an open position and the combiner partially extended between the stowed position and the display position.

FIG. 7 is a flow chart of a process by which the combiner positioning system operates, including locking the carrier in the display position and rotating the combiner in the display position.

FIG. 8 is a flow chart of a process by which the combiner positioning system operates, emphasizing the role of the camwheel in engaging the carrier and kinematically driving all sub-processes necessary to move the combiner between the stowed position and the display position.

### DETAILED DESCRIPTION OF EXEMPLARY EMBODIMENTS

Referring to FIG. 1, a HUD system 100 implemented in a vehicle is shown, according to an exemplary embodiment. The HUD system 100 may include a combiner 102 positioned between a driver of the vehicle and the windshield. Combiner 102 may protrude from slot 104 when in a display position and may retract into slot 104 when not in use.

HUD system 100 may include a combiner 102. Combiner 102 may be any type of combiner designed for operation in a HUD system or otherwise. For example, combiner 102 may be a translucent or transparent panel formed of a generally planar substrate including transparent or translucent polymers, glass, or other transparent or translucent material. Combiner 102 may be treated with one or more optical coatings selected to reflect specific wavelengths of monochromatic light. Combiner 102 may be a flat panel or may have a curvature.

Combiner 102 may be used in conjunction with a complete HUD system. In such a configuration, the HUD system may also include a light projector, a controller, or a video generator (not shown). The projector may be used to project visual images (e.g., pictures, graphics, video, text, etc.) received from a video generator onto combiner 102, which then reflects the visual images to a user. The projector may be located in the dashboard of the vehicle and oriented upward or at an angle to project light toward combiner 102. However, in other embodiments, the projector may be located elsewhere (e.g., attached to the roof, attached to a visor, etc.). Combiner 102 may be oriented vertically or at an angle such that light projected upon combiner 102 is reflected toward a user.

Still referring to FIG. 1, HUD system 100 may include a slot 104. Combiner 102 may protrude from slot 104 when in a display position as shown in FIG. 1. Additionally, combiner

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102 may retract into slot 104 for storage when not in use. Slot 104 may be a hole, slot, gap, space, or other opening through which combiner 102 may extend. Slot 104 may be an opening in an enclosure (e.g., a shell, a frame, a housing, a case, etc.) designed to contain combiner 102, or may exist in a separate component. A combiner positioning system may be used to transport combiner 102 between a display position and a stowed position.

Referring generally to FIG. 2A-FIG. 8, combiner positioning system 200 may move combiner 102 between the stowed position and the display position and rigidly hold combiner 102 in the display position, thereby reducing or eliminating the potential for vibration. Combiner positioning system 200 may use one or more camwheels in conjunction with one or more slots or pins to drive a carrier supporting combiner 102 between the stowed position and the display position. The camwheel(s) may be kinematically linked to a locking element, an angle adjustment support, and/or a flap. Advantageously, all components necessary to transport the combiner between the stowed position and the display position may be operated by the rotation of one or more axially linked camwheels.

Referring now to FIG. 2A and FIG. 2B, a combiner positioning system 200 is shown, according to an exemplary embodiment. FIG. 2A shows combiner positioning system 200 in a display position, whereas FIG. 2B shows combiner positioning system 200 in a stowed position. Referring specifically to FIG. 2A, combiner positioning system 200 may include a carrier 204 for supporting combiner 102, an axle 206 extending from carrier 204 about which carrier 204 and combiner 102 may rotate when in the display position, and a housing 220 for supporting and/or containing other components of system 200.

Still referring to FIG. 2A, combiner positioning system 200 is shown in a display position. Combiner positioning system 200 may include a carrier 204. Carrier 204 may be used to support combiner 102 and may be securely attached to combiner 102 to form a unit. Carrier 204 may extend along a length of combiner 102 and may be secured to combiner 102 by any suitable means (e.g., a clamp, a weld, a fastener, a clip, an adhesive compound, a screw, a bolt, etc.). For example, referring to FIG. 3, combiner 102 is shown fitted into a slot in carrier 204. However, in other embodiments, different or additional securing means may be used. In further embodiments, carrier 204 and combiner 102 may be formed into a single component such that no securing or attaching is required. Carrier 204 may have an upper portion for supporting combiner 102 and a lower portion for engaging a mechanism to transport carrier 204 between the stowed position and the display position.

Referring again to FIG. 2A, combiner positioning system 200 may further include an axle 206. Axle 206 may extend from carrier 204 or may be part of carrier 204. For example, in some embodiments, axle 206 may include two mirrored pins fitted into opposing sides of carrier 204, defining an axis extending through carrier 204 from one pin to the other pin. In other embodiments, axle 206 may include a single rod extending through carrier 204 along the axis. In further embodiments, axle 206 may be combined with carrier 204 or formed as part of a unitary component including both carrier 204 and axle 206. The precise structure of axle 206 is not limiting in this regard.

Still referring to FIG. 2A, in some embodiments, combiner positioning system 200 may include a housing 220. Housing 220 may be used to hold, contain, secure, protect, position, mount, fasten, or otherwise locate the various components of combiner positioning system 200. Housing 220 may be an

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enclosure, a frame, one or more surfaces, or any other means of orienting, protecting, containing, or positioning one or more elements of system 200. However, housing 220 is not required and should not be interpreted as a limitation on the scope of invention.

Referring now to FIG. 2B, combiner position system 200 is shown to further include one or more camwheels 230, rotatable about camwheel axis 232. Camwheels 230 may include a carrier pin 246 for engaging carrier 204 and an angle adjustment support 234. Combiner positioning system 200 may rotate camwheel 230 and engage carrier 204, thereby moving carrier 204 and combiner 102 between the stowed position and the display position. Angle adjustment support 234 may be used to adjust the angle of combiner 102 or support carrier 204 when in the display position.

Still referring to FIG. 2B, positioning system 200 may include a camwheel 230. Camwheel 230 may be rotatable about a camwheel axis 232. In some embodiments, rotation of camwheel 230 about camwheel axis 232 may be accomplished automatically by any type of motor, drive train system, or power system. In other embodiments, camwheel 230 may be manually rotated either directly or via an intermediate means of interaction (e.g., a handle, a crank, a dial, a wheel, etc.). Camwheel 230 may be any type of material including metals, polymers, woods, natural or synthetic compounds, or any other suitable material. Camwheel 230 is shown to be generally cylindrical in shape, wherein the radius of the camwheel cylinder exceeds the length of the cylinder. However, other camwheel shapes may be used (e.g., square, rectangular, triangular, irregular shape, etc.).

In some embodiments, camwheel 230 may be positioned to one side of combiner 102 (e.g., the left side or right side). However, in other embodiments, multiple camwheels 230 may be used. For example, one camwheel may be positioned on one side of combiner 102 and another camwheel may be positioned on the other side of combiner 102. If multiple camwheels are used, the camwheels may be connected by a shared camwheel axle extending from one camwheel to another camwheel along camwheel axis 232. Thus, rotation of one camwheel or the camwheel axle may cause the other camwheel(s) to rotate. Additionally, if multiple camwheels are used, various components of camwheel positioning system 200 may be positioned relative to one or more of the camwheels, as described below.

Referring now to FIG. 3, in some embodiments, combiner positioning system 200 may include a guide element 208. Guide element 208 may define a path along which axle 206 moves between a display position and a stowed position. In some embodiments, guide element 208 may be a slot or channel within which axle 206 is constrained. In other embodiments, guide element 208 may be a positive extrusion, protrusion, rail, or other guiding element capable of guiding axle 206 between the stowed position and the display position. For example, guide element 208 may be formed into housing 220 as a slot, as shown in FIG. 3. However, guide element 208 is optional and is not intended to limit the scope of invention unless recited in the claims.

Referring now to FIG. 4A-FIG. 4D, combiner positioning system 200 is shown to further include a locking element 210. FIG. 4A and FIG. 4B show locking element 210 in a disengaged position and FIG. 4C and FIG. 4D show locking element 210 in an engaged position. In an exemplary embodiment, locking element 210 may engage axle 206 when combiner 102 is in the display position. Locking element 210 may rigidly hold axle 206 in the display position, thereby reducing the potential for vibration and providing a stable

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axis about which carrier **204** and combiner **102** may rotate to adjust the display angle of combiner **102**.

Locking element **210** may increase the resonance frequency or natural frequency of combiner **102**, carrier **204**, or combiner positioning system **200** as a whole by restraining axle **206** from moving laterally (e.g., movement along guide element **208** or otherwise) and by providing increased stiffness (e.g., resistance to movement, rigidity, stability, etc.) when locking element **210** engages carrier **204** in the display position. However, locking element **210** does not prevent carrier **204** and combiner **102** from rotating about an axis defined by axle **206** when in the display position. This allows the display angle of combiner **102** to be adjusted in the display position.

In some embodiments, locking element **210** may be shaped as a lever having a lever axle **214**. Locking element **210** may rotate about lever axle **214** between a disengaged position (shown in FIG. 4A and FIG. 4B) and an engaged position (shown in FIG. 4C and FIG. 4D). However, in other embodiments, locking element **210** may be shaped as a slider, a clip, a ledge, or any other element capable of securing axle **206** in the display position. When locking element **210** is in the disengaged position, axle **206** may freely move between the display position and the stowed position. However, when locking element **210** is in the engaged position, axle **206** is restrained from lateral movement and is only allowed to rotate.

Referring now to FIG. 4E, in some embodiments, locking element **210** may urge axle **206** into a recess **212** located at one end of guide element **208**. Recess **212** may provide a secure resting position for axle **206** in the display position. In some embodiments, recess **212** may be a separate fixed or movable part mounted or otherwise rigidly attached to housing **220**. In further embodiments, recess **212** may be incorporated into guide element **208** (e.g., an end of guide element **208** may define recess **212**) or may be eliminated entirely. Advantageously, recess **212** may be designed to allow carrier **204** and combiner **102** to be removed from housing **220** for repair or replacement without requiring disassembly of the entire combiner positioning system **200**.

In some embodiments, a spring (not shown) may be attached to locking element **210**. The spring may provide increased resilience for holding axle **206** in the display position when engaged by locking element **210**. In some embodiments, the spring may be located on the opposite side of locking element **210** from axle **206**, thereby providing a compression force urging locking element **210** and axle **206** toward recess **212**. However, in other embodiments, the spring may be located elsewhere (e.g., above recess **212**, on the other side of housing **220**, etc.). In some embodiments, the spring may exert a tension force rather than a compression force, depending on its location, or may be a rotational spring exerting a torque rather than a force. However, in further embodiments, the spring may be eliminated entirely.

Referring again to FIG. 4A, camwheel **230** may contain a locking element slot **236**. Locking element slot **236** may be located in one or more camwheel **230** and may be placed on either side of camwheel **230**.

Referring still to FIG. 4A, locking element **210** may be coupled to camwheel **230** via a locking element pin **238**. Locking element pin **238** may extend from locking element **210** and may be fitted into locking element slot **236** in camwheel **230**. Locking element pin **238** and locking element slot **236** may be positioned on either side of camwheel **230**. Locking element slot **236** may be substantially circumferentially oriented relative to camwheel **230**. In some embodiments, the radial distance between locking element slot **236** and cam-

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wheel axis **232** may vary along the length of the slot **236**. Thus, rotation of camwheel **230** may cause locking element **210** to rotate about lever axle **214**, thereby moving locking element **210** between the engaged position and disengaged position.

Advantageously, locking element slot **236** may be designed such that locking element **210** engages axle **206** or carrier **204** only when axle **206** is in the display position. Therefore, it may be kinematically impossible for locking element **210** to prematurely move into the engaged position before axle **206** or carrier **204** has reached the display position.

Referring now to FIG. 5A and FIG. 5B, a side sectional view of combiner positioning system **200** is shown. Referring specifically to FIG. 5A, camwheel **230** may include a carrier pin **246** extending from the camwheel **230**. As camwheel **230** rotates, carrier pin **246** may engage carrier **204**. As camwheel **230** continues to rotate, carrier **204** may be moved between the stowed position (shown in FIG. 5A) and the display position (shown in FIG. 5B).

Referring specifically to FIG. 5B, in some embodiments, carrier pin **246** may disengage from carrier **204** once carrier **204** has reached the display position. By disengaging carrier pin **246** from carrier **204**, camwheel **230** may further rotate without moving carrier **204** from the display position.

Referring now to FIG. 5C, camwheel **230** may include an angle adjustment support **234**. Angle adjustment support **234** may extend from camwheel **230** and may be generally circumferentially oriented around camwheel axis **232**. As camwheel **230** continues to rotate, angle adjustment support **234** may engage carrier **204** in the display position or prior to reaching the display position. In some embodiments, the radial distance between camwheel axis **232** and angle adjustment support **234** may vary along the circumferential length of angle adjustment support **234**. Thus, further rotation of camwheel **230** once angle adjustment support **234** has engaged carrier **204** may cause carrier **204** and combiner **102** to rotate about axle **206**.

For example, still referring to FIG. 5C, angle adjustment support **234** is shown engaging carrier **204**. As camwheel **230** continues to rotate, angle adjustment support **234** may move from the position shown in FIG. 5C to the position shown in FIG. 5D. Therefore, angle adjustment support **234** may be used to adjust the angle of combiner **102** when in the display position in order to more accurately reflect projected light toward a user.

Still referring to FIG. 5C, in some embodiments, angle adjustment support **234** may be a self-locking element. For example, as camwheel **230** rotates, angle adjustment support **234** may urge, press, or force carrier **204** into the display position due to the varying radial distance between angle adjustment support **234** and camwheel axis **232**. Once in the display position, some or all of the load (e.g., weight, lifting force, torque) necessary to hold carrier **204** in the display position may be transferred to angle adjustment support **234**. Angle adjustment support **234** may rigidly hold carrier **204** in the display position and may be reinforced with ridges **252** extending radially between angle adjustment support **234** and camwheel axis **232**. Reinforcing ridges **252** may stiffen the connection between angle adjustment support **234** and camwheel **230**, thereby increasing the resonance frequency or natural frequency of combiner **102**, carrier **204**, or combiner positioning system **200** as a whole.

Referring now to FIG. 6A and FIG. 6B, combiner positioning system **200** may include a flap **222**. FIG. 6A shows flap **222** in a closed position whereas FIG. 6B shows flap **222** in an open position. Flap **222** may cover a slot or gap (e.g., slot **104**)

if combiner positioning system **200** is contained within a housing. Flap **222** may be made of any material, including polymers, metals, woods, glass, synthetic or natural compounds, etc. Flap **222** may be rigid, flexible, semi-flexible, or have any intermediate degree of flexibility. Flap **222** may be opaque, translucent, or transparent and may have any color or visual appearance.

In some embodiments, flap **222** may move between an open position and a closed position. For example, in the closed position, flap **222** may cover slot **104**, thereby protecting combiner **102** and the other components of combiner positioning system **200** from environmental damage (e.g., pollution, foreign particles, excessive heat, physical, electrical, or chemical damage, etc.). In the open position, flap **222** may retract from, or otherwise expose slot **104** such that combiner **102** may extend through slot **104** into the display position.

Referring again to FIG. 4A, flap **222** may be coupled to camwheel **230** via a flap pin **242**. Flap pin **242** may extend from flap **222** and may be fitted into a flap slot **240** in camwheel **230**. Flap pin **242** and flap slot **240** may be positioned on either side of camwheel **230**. Flap slot **240** may be substantially circumferentially oriented relative to camwheel **230**. In some embodiments, the radial distance between flap slot **240** and camwheel axis **232** may vary along the length of the slot **240**. Thus, rotation of camwheel **230** may cause flap pin **242** to move relative to camwheel axis **232** and thereby cause flap **222** to retract from or otherwise expose slot **104**.

Advantageously, flap slot **240** may be designed (e.g., incorporated into camwheel **230**) such that flap **222** is withdrawn from slot **104** via rotation of camwheel **230** before combiner **102** is moved into the space previously occupied by flap **222** when moving combiner **102** into the display position. Therefore, it may be kinematically impossible for combiner **102** to contact flap **222** or to extend into the display position before flap **222** has been withdrawn from slot **104**.

In some embodiments, camwheel **230** may be positioned to one side of combiner **102** (e.g., the left side or right side). However, in other embodiments, multiple camwheels may be used. For example, one camwheel may be positioned on one side of combiner **102** and another camwheel may be positioned on the other side of combiner **102**. If multiple camwheels are used, the camwheels may be connected by a shared camwheel axle extending from one camwheel to another camwheel along camwheel axis **232**. Thus, rotation of one camwheel or the camwheel axle may cause the other camwheel(s) to rotate.

Additionally, if multiple camwheels are used, the various components of camwheel positioning system **200** (e.g., pins **238**, **242**, and **246**, slots **236** and **240**, locking element **210**, angle adjustment support **234**, guide element **208**) may be coupled to one or more of the camwheels. For example, locking element **210** may be located on only one side of combiner **102** and may be rotated via locking element slot **236** in the camwheel on that side. On the other side of combiner **102**, angle adjustment support **234** may be used in place of, or in addition to, locking element **210** and locking element slot **236**.

Referring to FIG. 7, a flow chart of a process **700** by which combiner positioning system **200** operates is shown, according to an exemplary embodiment. In brief overview, process **700** may be used to move the combiner between the stowed position and the display position (step **702**) and engage a portion of the carrier with a locking element and hold the carrier in the display position (step **704**). Optionally, process **700** may also be used to rotate the carrier about a carrier axis (step **706**), disengage the portion of the carrier with the lock-

ing element (step **708**), and move the carrier from the display position to the stowed position (step **710**). Steps shown with broken lines around the edges (e.g., step **706-710**) are optional depending on the particular configuration of the combiner positioning system.

Still referring to FIG. 7, process **700** is shown to include moving a carrier between a stowed position and a display position (step **702**). In some embodiments, axle **206** may extend from carrier **204** and step **702** may be accomplished by moving axle **206** between the stowed position and display position. In some embodiments, step **702** may include moving axle **206** along guide element **208**, which may be fastened to or embedded in a housing **220**. However, in other embodiments, guide element **208** and housing **220** are unnecessary and may be eliminated. Step **702** may be accomplished using a drive motor or camwheel system to move carrier **204** between the stowed position and the display position. However, in other embodiments, other power systems may be used.

Still referring to FIG. 7, process **700** is shown to further include engaging a portion of the carrier with a locking element and rigidly holding the carrier in the display position (step **704**). In some embodiments, step **704** may be accomplished using locking element **210** to engage axle **206** when in the display position and rigidly hold axle **206** in place. In some embodiments, locking element **210** may urge axle **206** into a recess **212** located at one end of guide element **208**. However, in other embodiments, recess **212** and guide element **208** are unnecessary. Step **704** may be accomplished using a drive motor or camwheel system to engage the portion of the carrier with the locking element upon reaching the display position. However, in other embodiments, other power systems may be used.

Process **700** is shown to optionally include rotating the carrier and combiner about an axis defined by an axle extending from the carrier (step **706**). Step **706** may be performed to adjust the angle of combiner **102** when in the display position in order to more accurately reflect projected light toward a user. Step **706** may be accomplished using a drive motor or camwheel system to rotate carrier **204** and combiner **102** about axle **206**. However, in other embodiments, other power systems may be used.

Process **700** is further shown to optionally include disengaging the portion of the carrier with the locking element (step **708**) and moving the carrier from the display position to the stowed position (step **710**). Steps **708** and **710** may be performed to protect combiner **102** by retracting combiner **102** into housing **220** or another protective component in order to prevent damage to combiner **102** or combiner positioning system **200** when the HUD system is not in use. Step **708** may be accomplished by moving locking element **210** from an engaged position to a disengaged position, thereby allowing axle **206** to move laterally along guiderail **208** or otherwise. If step **706** is optionally performed, it may be necessary to rotate carrier **204** and combiner **102** such that axis **206** may be moved laterally. Then carrier **204** and combiner **102** may be moved from the display position to the stowed position. Step **708** and step **710** may be accomplished using a drive motor or camwheel system to disengage locking element **210**, rotate combiner **102** and carrier **204**, and move combiner **102** and carrier **204** from the display position to the stowed position. However, in other embodiments, other power systems may be used.

Referring now to FIG. 8, a flow chart of a process **800** by which combiner positioning system **200** may operate is shown, according to an exemplary embodiment. In brief overview, process **800** may be used to move the combiner between

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the stowed position and the display position (step 808), engage an axle extending from the carrier with a locking element (step 810), engage the carrier with an angle adjustment support (step 814), and rotate the carrier about a carrier axis (step 816). Steps shown with broken lines around the edges (e.g., step 806, and 810-816) are optional depending on the particular configuration of the combiner positioning system. Advantageously, all of steps 804-816 are accomplished by rotating camwheel 230 about a camwheel axis (step 802). In other words, step 802 kinematically drives all other steps of process 800.

For example, rotating camwheel 230 (step 802) may cause carrier pin 246 to engage carrier 204 (step 804) and move carrier 204 along with combiner 102 between the stowed position and the display position (step 808). Optionally, if flap 222 is used in combiner positioning system 200, rotating camwheel 230 may also cause flap 222 to move between an open position and closed position (step 806) prior to moving combiner 102 into the display position (step 808). However, if combiner 102 is being moved from the display position to the stowed position, step 806 would occur after step 808, as the process would be performed in reverse order.

Still referring to FIG. 8, process 800 may include one or more of optional steps 810-814, all of which are also accomplished by continuing to rotate camwheel 230. For example, process 800 may include engaging an axle 206 extending from the carrier 204 with a locking element 210 (step 810). Continued rotation of camwheel 230 may accomplish step 810 by rotating locking element 210 between a disengaged position (shown in FIGS. 4A-4B) and an engaged position (shown in FIGS. 4C-4D) as locking element pin 238 slides along locking element slot 236.

Locking element 210 may discharge the force exerted on carrier pin 246, thereby allowing carrier pin 246 to disengage from carrier 204 (step 812). Once carrier pin 246 has disengaged, camwheel 230 may continue to rotate without affecting the lateral position of carrier 204.

Process 800 may also include engaging the carrier 204 with an angle adjustment support 234 (step 814) as shown in FIG. 5C and FIG. 5D. Angle adjustment support 234 may perform a similar function to locking element 210 by restraining carrier 204 and combiner 102 in the display position (e.g., preventing lateral movement). Additionally, due to the varying radial distance between angle adjustment support 234 and camwheel axis 232 (i.e., one end of angle adjustment support 234 may be closer to axis 232 than the other end), angle adjustment support 234 may cause carrier 204 and combiner 102 to rotate about a carrier axis (step 816) as camwheel 230 continues to rotate.

Advantageously, all steps of process 800 are driven by the rotation of camwheel 230. By rotating camwheel 230 in one direction, all steps necessary to move combiner 102 from the stowed position to the display position, and also several optional steps, are kinematically carried out. Because all moving parts (e.g., pins, levers, support structures, etc.) are directly connected to one or more camwheels, no intermediate elements (e.g., additional gears, wheels, levers, etc.) are required. In other words, all movement, whether rotational, linear, or a combination thereof, are kinematically caused by the rotation of one or more axially coupled camwheels.

The construction and arrangement of the systems and methods as shown in the various exemplary embodiments are illustrative only. Although only a few embodiments have been described in detail in this disclosure, many modifications are possible (e.g., variations in sizes, dimensions, structures, shapes and proportions of the various elements, values of parameters, mounting arrangements, use of materials, colors,

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orientations, etc.). For example, the position of elements may be reversed or otherwise varied and the nature or number of discrete elements or positions may be altered or varied. Accordingly, all such modifications are intended to be included within the scope of the present disclosure. The order or sequence of any process or method steps may be varied or re-sequenced according to alternative embodiments. Other substitutions, modifications, changes, and omissions may be made in the design, operating conditions and arrangement of the exemplary embodiments without departing from the scope of the present disclosure.

Although the figures may show a specific order of method steps, the order of the steps may differ from what is depicted. Also two or more steps may be performed concurrently or with partial concurrence. All such variations are within the scope of the disclosure.

The invention claimed is:

1. A combiner positioning system for a head-up display, the system comprising:

a carrier supporting a combiner for the head-up display, wherein the combiner is movable between a stowed position and a display position;

a camwheel rotatable about a camwheel axis; and a pin extending from the camwheel,

wherein rotation of the camwheel about the camwheel axis engages the pin with the carrier and moves the combiner between the stowed position and the display position, and wherein the pin disengages from the carrier when in the display position such that further rotation of the camwheel rotates the carrier about a carrier axis but does not cause further lateral movement of the carrier, wherein lateral movement is required for the carrier to move between the stowed position and the display position.

2. The combiner positioning system of claim 1, wherein the camwheel engages the carrier directly without any intermediate components.

3. The combiner positioning system of claim 1, further comprising an angle adjustment support extending from the camwheel, wherein rotation of the camwheel about the camwheel axis engages the carrier with the angle adjustment support and rotates the combiner about a carrier axis, wherein the angle adjustment support is a self-locking element.

4. The combiner positioning system of claim 3, wherein the angle adjustment support rigidly holds the carrier in the display position and increases a stiffness of the combiner positioning system when holding the carrier in the display position.

5. The combiner positioning system of claim 1, further comprising a locking element having an engaged position and a disengaged position, wherein the locking element engages an axle extending from the carrier in the display position and rigidly holds the axle in the display position.

6. The combiner positioning system of claim 5, wherein, when the locking element is in the disengaged position, the locking element does not engage the axle and does not restrain the axle from moving laterally, wherein lateral movement of the axle is required for the combiner to move between the stowed position and the display position.

7. The combiner positioning system of claim 5, wherein the locking element is shaped as a lever rotatable about a lever axis between the engaged position and the disengaged position,

wherein the locking element is coupled to the camwheel via a pin extending from the locking element and fitted into a slot in the camwheel, wherein rotation of the camwheel about the camwheel axis rotates the locking



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element about the lever axis between the engaged position and the disengaged position.

8. The combiner positioning system of claim 1, further comprising a guide element defining a path along which an axle extending from the carrier moves between the stowed position and the display position.

9. The combiner positioning system of claim 1, further comprising a housing within which the combiner is contained when in the stowed position and from which the combiner protrudes when in the display position.

10. The combiner positioning system of claim 9, further comprising a flap movable between an open position and a closed position, wherein the flap covers an opening in the housing when in the closed position and exposes the opening when in the open position,

wherein the flap is coupled to the camwheel via a pin extending from the flap and fitted into a slot in the camwheel, wherein rotation of the camwheel about the camwheel axis moves the flap between the open position and the closed position.

11. A method of positioning a combiner for a head-up display, the method comprising:

rotating a camwheel about a camwheel axis;

engaging a carrier supporting a combiner for a head-up display;

moving the combiner between a stowed position and a display position, wherein rotation of the camwheel about the camwheel axis engages the carrier and moves the combiner between the stowed position and the display position; and

disengaging the carrier from a pin extending from the camwheel when in the display position such that further rotation of the camwheel does not cause further lateral movement of the carrier once in the display position, wherein lateral movement is required for the carrier to move between the stowed position and the display position.

12. The method of claim 11, wherein the camwheel engages the carrier directly without any intermediate components.

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13. The method of claim 11, further comprising: engaging the carrier with an angle adjustment support extending from the camwheel, wherein the angle adjustment support engages the carrier when in the display position; and

rotating the carrier about a carrier axis, wherein further rotation of the camwheel about the camwheel axis engages the carrier with the angle adjustment support and rotates the carrier about the carrier axis via the angle adjustment support.

14. The method of claim 13, further comprising: rigidly holding the carrier in the display position with the angle adjustment support.

15. The method of claim 11, further comprising: engaging an axle extending from the carrier with a locking element, wherein the locking element has an engaged position and a disengaged position; and rigidly holding the axle in the display position with the locking element,

wherein when the locking element is in the disengaged position, the locking element does not engage the axle and does not restrain the axle from moving laterally, wherein lateral movement of the axle is required for the combiner to move between the stowed position and the display position.

16. The method of claim 15, wherein the locking element is shaped as a lever rotatable about a lever axis between the engaged position and the disengaged position, wherein the locking element is coupled to the camwheel such that rotation of the camwheel about the camwheel axis rotates the locking element about the lever axis between the engaged position and the disengaged position.

17. The method of claim 11, wherein the combiner is contained within a housing when in the stowed position and protrudes from the housing when in the display position.

18. The method of claim 17, further comprising: moving a flap between an open position and a closed position, wherein the flap covers an opening in the housing when in the closed position and exposes the opening when in the open position,

wherein the flap is coupled to the camwheel such that rotation of the camwheel about the camwheel axis moves the flap between the open position and the closed position.

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